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"DESIGN & STATIC ANALYSIS OF SPUR GEAR USING DIFFERENT MATERIAL BY

ANSYS SOFTWARE"

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ABSTRACT

In the gear design the bending stress and surface strength of the gear tooth are considered to be one of the main contributors for the failure of the gear in a gear set. Thus, the analysis of stresses has become popular as an area of research on gears to minimize or to reduce the failures and for optimal design of gears In this paper contact stresses are calculated by using analytical method as well as Finite element analysis. To estimate bending stress modified Lewis beam strength method is used. CATIA solid modeling software is used to generate the 3-D solid model of spur gear. ANSYS workbench software package is used to analyze the bending stress. Contact stresses are calculated by using modified AGMA contact stress method. In this also CATIA modeling software is used to generate contact gear tooth model. ANSYS software package is used to analyze the contact stress. Finally these two methods contact stress results are compared with each other.

Keyword: CATIA, ANSYS, Contact stress, Gear, Spur gear, FE method

I. INTRODUCTION

Gears are used for a wide range of industrial applications. They have varied application starting from textile looms to aviation industries. They are the most common means of transmitting power. They change the rate of rotation of machinery shaft and also the axis of rotation. For high speed machinery, such as an automobile transmission, they are the optimal medium for low energy loss and high accuracy. Their function is to convert input provided by prime mover into an output with lower speed and corresponding higher torque. Toothed gears are used to transmit the power with high velocity ratio. During this phase, they encounter high stress at the point of contact. A pair of teeth in action is generally subjected to two types of cyclic stresses:

- i) Bending stresses inducing bending fatigue
- ii) Contact stress causing contact fatigue.





Fig 1.1 Spur Gear

Spur gears are the most common type of gears. They are used to transmit rotary motion between parallel shafts i.e., they are usually cylindrical in shape, and the teeth are straight and parallel to the axis of rotation. Sometimes many spur gears are used at once to create very large gear reductions. Spur gears are used in many devices but not in cars as they produce large noises.

II. PROBLEM DEFINITION

One of the main causes for failure of the gear tooth is bending stresses near the root of the gear and the contact stresses where the gears meet. The main objective of this paper is to analyze the bending stresses in the spur gear. When the spur gears mesh a tangential and a radial load acts upon the gear tooth and this generates stresses in the gear tooth. The radial load induces compressive stress of relatively small magnitude therefore its effect on the tooth may be neglected. The tangential load induces a bending stress which tends to break the tooth.

Failure by bending will occur when the significant tooth stress equals or exceeds either the yield strength or the bending endurance strength of the material. This paper investigates bending stress developed in gear set while transmitting power for both the steel and Aluminium as gear material. Both above said material find many applications and also each material exhibits their own characteristics during service condition, high strength, durability and load carrying capacity creates an opportunities to use Steel as gear material and in contrast aluminium as a gear material shows up unique characteristics like corrosion resistance, light weight and easy of machining.

III. SPECIFICATION OF THE GEAR

Parameters	Symbols	Unit	Value
Number of teeth	Z		40
Module	m	mm	6
Power	Р	kW	1500
Speed	Ν	RPM	1500
Pitch circle diameter	D	mm	250
Pressure Angle	α	Degree	14.5
Face width	b	mm	22

Table. 3.1: Specifications of Gear.

IV. PROPERTIES OF MATERIALS USED FOR BOTH PINION AND GEAR

Material Field Variable	Value	Units
Density	7750	Kg/m ³
Young's modulus	1.93E+05	Мра
Poisson Ratio	0.31	
Shear modulus	76664	Mpa
Bulk Modulus	1.6937E+05	Mpa
Tensile Yield Strength	207	Mpa
Compressive Yield Strength	207	Мра
Tensile Ultimate Strength	586	Мра

I. Table 4.1 Structure Steel Mechanical properties

II. Table 4.2 Aluminium Alloy Mechanical properties

Material Field	Value	Units
Variable		
Density	2770	Kg/m ³
V	7.15.10	Max
Young's modulus	/.1E+10	мра
Poisson Ratio	0.33	
1 0135011 14110	0.55	
Shear modulus	2.6692E+04	MPa
Bulk Modulus	6.9608E+04	MPa
Tensile Yield Strength	280	Мра
<u> </u>	200	
Compressive Yield	280	Мра
Strength		
Tensile Ultimate	310	Mpa
Strength		



Material Field	Value	Units
Variable		
Density	7200	Kg/m ³
Young's modulus	1.11E+05	Мра
Poisson Ratio	0.28	
Shear modulus	42969	Мра
Bulk Modulus	83333	Mpa
Tensile Yield Strength	240	Mpa
Tensile Ultimate Strength	820	Мра

III. Table 4.3 Cast Iron Alloy Mechanical properties

IV. Table 4.4 Carbon Fiber Mechanical properties

	V. I.	T I
Material Field	value	Units
Variable		
Density	1950	Kg/m ³
Young's modulus	300000	Мра
Poisson Ratio	0.3	
Shear modulus	1.15E+05	Мра
Bulk Modulus	2.5E+05	Mpa
Tensile Strength	5090	Мра
Compressive Strength	1793	Мра



V. THEORETICAL STRESS CALCULATION

Lewis Equation is used in order to calculate the theoretical bending stresses. Lewis considered the gear tooth as a cantilever beam which is loaded at its free end. Lewis form factor is given by Pitch line velocity

$$\begin{split} V &= \Pi \ D_p N_p / 60 \ = \Pi \ x \ 0.125x \ 1500 / 60 \ = 9.81 \ m/s \\ \text{Lewis equations } 14.5 \ \text{involute teeth}, \ \text{then Tooth form factor} \\ Y_G &= 0.124 - 0.684 / T_p \ = 0.124 - 0.684 / 20 \ = 0.089 \\ \text{Ordinary cut gears and operating at velocity ratio is up to } 12.5 \ m/s \\ C_v &= 3/3 + v \\ C_v &= 3/3 + 9.81 \ = 0.234 \\ \text{Design Tooth load} \\ W_T &= P \ Cs/v \\ W_T &= 15000 \ x1 \ / 9.81 \ = 1529.1 \ N \\ W_T &= 6_w.b.pc.y \ = \ 6_w.b.\Pi \ m.y \ = (6_o.Cv) \ b.\Pi m.y \\ \sigma_w. &= W_T \ / \ b.\Pi m.y \\ \sigma_w. &= 116.5 \ \text{MPa} \\ \\ \text{Theoretical bending stress of the designed gear = } 116.5 \ \text{Mpa} \end{split}$$

VI. BENDING STRESS CALCULATION USING ANSYS

The gear is first designed in the ANSYS designer workbench. The calculated co-ordinates are plotted and an involute profile is generated. The addendum circle and dedendum circles are drawn and the profile is connected to for the individual gear tooth.



Fig. 6.1 Geometry of Spur gear for Structural Steel

The modeled gear is then imported into ANSYS STATIC STUCTURAL analysis workbench. The two gears are then meshed and the mesh near the root and the contact surfaces between the gears are refined in order to get more accurate



bending stress values. The bending stresses are analyzed by fixing one of the gear using fixed support and the other gear with the frictionless support. The gear with the frictionless support is then applied a moment of 96936 N-mm.



Fig.6.2 Boundary Condition applied moments on Structural Steel Gears



Fig. 6.3 Equivalent Von misses Stress for Structural Steel





Fig. 6.4 Total Deformation on Spur gears for Structure steel



Fig. 6.5 Equivalent Von misses Stress for Cast Iron Alloy



Fig. 6.6 Total Deformation on Spur gears for Cast Iron Alloy



Fig. 6.7 Equivalent Stresses on Spur gears for Aluminium Alloy



Fig. 6.8 Total Deformations on Spur gears for Aluminium Alloy



Fig. 6.9 Equivalent Stresses on Spur gears for Carbon Fiber http://www.ijrtsm.com© International Journal of Recent Technology Science & Management





Fig. 6.10 Equivalent Stresses on Spur gears for Carbon Fiber

VII. RESULT AND DISCUSSIONS

In this work we find value of vonmisses stresses, Aluminium Alloy, Structural Steel, Carbon Fiber and Cast Iron Alloy are respectively 59.975 MPa, 60.003MPa, 60.003MPa and 60.028 MPa And total deformation for these materials likes Carbon Fiber, Structural Steel, Aluminium Alloy and Cast Iron Alloy and, are respectively 0.018914 mm, 0.039769 mm, 0.50603 mm, and 0.09732 mm. Here we can see that we have used four different materials in all materials we will be selected composite material to other than because it is light weight and heavy duty its deformation and stresses range are considerable.



Fig.7.1 Frictional Von misses comparison graph for different materials



Fig.7.2 Deformations comparison graph for different materials

VIII. CONCLUSION

In this work analytical and Finite Element Analysis methods were used to predicting the Bending stresses of involute Spur gear. Bending stresses are calculated by using AGMA and ANSYS software package. Bending stresses are calculated by using ANSYS software package.

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